



For two-letter codes and other abbreviations, refer to the "Guidance Notes on Codes and Abbreviations" appearing at the beginning of each regular issue of the PCT Gazette.

DESCRIPTION

Electric Compressor

5 Technical Field

The present invention relates to a lubricating mechanism of an electric compressor to be used in cooling devices such as a refrigerator.

Background Art

10 In general, an electric compressor has a lubricating mechanism at its shaft, and Japanese Patent Examined Publication No. S62 - 44108 discloses one of those instances. Fig. 5 shows a sectional view of this conventional compressor, and Fig. 6 shows an electric connection diagram of this compressor.

In Fig. 5, hermetic container 1 accommodates electric motor 4 formed of
15 stator 18 and rotor 8, and compressing mechanism 2. Shaft 7 extends through bearing 6 of block 3, and rotor 8 of the motor is rigidly mounted to an outer wall of shaft 7, of which eccentric shaft 9 is coupled to piston 10 by slider 11. Shaft 7 includes centrifugal pump 12 formed at its lower end and opening into lubricant 17.

20 Shaft 7 includes spiral groove 14, engraved on its outer wall and having a lead, for leading lubricant 17 upward when the motor rotates in a predetermined forward direction. A lower end of spiral groove 14 communicates with centrifugal pump 12, and an upper end of spiral groove 14 communicates with annular lubricant groove 16 (not shown) formed on an
25 upper end of bearing 6.

A lower end of vertical hole 15 bored in eccentric shaft 9 communicates with the annular lubricant groove 16, and an upper end of hole 15 opens into a

space of hermetic container 1.

As shown in Fig. 6, stator 18 of the motor includes main coil 19 and starting coil 20. PTC (Positive Temperature Co-efficient) relay 21 is coupled to starting coil 20 in series, so that a resistance-start type of single-phase
5 induction motor is formed.

Application of a voltage starts the motor rotating in a forward direction, and a temperature of elements of PTC relay 21 sharply rises, which accompanies a sharp increase in the resistance of the elements, so that starting coil 20 is actually cut off, and the motor is driven only by main coil 19.
10 Lubricant 17 is sucked up to spiral groove 14 by centrifugal pump 12, and rotation of spiral groove 14 transports lubricant 17 upward for lubricating sliding sections of the compressor.

However, since the conventional electric compressor discussed above prepares the winding direction of the lead of the spiral groove 14 based on an
15 assumption of a forward rotating direction, spiral groove 14 fails to transport the lubricant upward if the motor rotates in a reverse direction due to some reason. As a result, the sliding sections encounter no lubricant. This reverse rotation lasts until the compressor is stopped (max. several hours), and the motor returns to the forward rotation when the motor is re-started. However,
20 abrasion sometimes occurs in the sliding sections during the reverse rotation.

Disclosure of the Invention

The present invention addresses the problem discussed above, and aims to provide an electric compressor that can lubricate the sliding sections with a
25 minimum quantity even if the motor rotates in a reverse direction.

The electric compressor of the present invention includes a shaft having a forward leading groove and a reverse leading groove both engraved on its outer

wall. The forward leading groove transports lubricant upward for lubricating sliding sections when the motor rotates in a forward direction. The reverse leading groove has a lead directed oppositely to that of the forward leading groove, and transports the lubricant upward for lubricating the sliding sections
5 when the motor rotates in a reverse direction.

Brief Description of the Drawings

Fig. 1 is a sectional view of an electric compressor in accordance with an exemplary embodiment of the present invention.

10 Fig. 2 is an enlarged view of a shaft of the compressor shown in Fig. 1.

Fig. 3 is an enlarged view of a shaft of the compressor shown in Fig. 1.

Fig. 4 is an electric connection diagram of a motor of the compressor shown in Fig. 1.

Fig. 5 is a sectional view of a conventional compressor.

15 Fig. 6 is an electric connection diagram of a motor of the conventional compressor.

Detailed Description of Preferred Embodiment

An exemplary embodiment of the present invention is demonstrated hereinafter with reference to the accompanying drawings. Fig. 1 is a sectional view of an electric compressor in accordance with an exemplary embodiment of the present invention. Fig. 2 and Fig. 3 show enlarged views of a shaft of the compressor shown in Fig. 1. Fig. 4 is an electric connection diagram of a motor of the compressor.
20

25 In Figs. 1, 2, and 3, lubricant 103 is pooled in hermetic container 101. Compressing mechanism 111 is disposed on an upper section of single-phase induction motor 109 that is formed of stator 105 and rotor 107. Compressing

mechanism 111 is resiliently supported by spring 115 via stator 105 and accommodated in hermetic container 101.

Bearing 121 is formed in block 119. Shaft 127 having main shaft 123 and sub-shaft 125 penetrates through bearing 121, and rotor 107 is rigidly
5 mounted to main shaft 123. Piston 129 reciprocally penetrates through cylinder 117 disposed in block 119. Sub-shaft 125 is coupled with piston 129 by connecting rod 131.

Centrifugal pump 133 is formed at a lower end of main shaft 123, and opens into lubricant 103. A thinner section 135 having a smaller diameter
10 than that of main shaft 123 is formed at a part of main shaft 123. Forward leading groove 137 and reverse leading groove 139, having a lead directed oppositely to that of forward leading groove 137, are engraved on the outer wall of main shaft 123. Entire rounding section of the upper end of bearing 121 is chamfered, and annular lubricant groove 141 is formed between the chamfered
15 section and main shaft 123.

A first end of forward leading groove 137 communicates with centrifugal pump 133, and a second end thereof opens directly to annular lubricant groove 141. A first end of reverse leading groove 139 communicates with centrifugal pump 133 via thinner section 135, and a second end thereof directly opens to
20 annular lubricant groove 141. A cross sectional area of reverse leading groove 139 is smaller than that of forward leading groove 137, and the lead of reverse leading groove 139 is greater than that of forward leading groove 137.

Vertical hole 143, of which first end communicates with annular lubricant groove 141 and second end opens in hermetic container 101, is
25 provided in sub-shaft 125. Vertical hole 143 slants with respect to the center of shaft 127 such that its upper section slants outward.

As shown in Fig. 4, stator 105 includes main coil 145 and starting coil

147. PTC relay 149 to be used for starting the motor is coupled to starting coil 147 in series.

An operation and an effort of the compressor having the structure discussed above is demonstrated hereinafter. First, an AC power supply is applied to the motor, and a current runs through main coil 145 and starting coil 147, so that rotor 107 starts rotating in a predetermined forward direction. Then PTC relay 149 increases resistance sharply at its elements, so that the current supply to starting coil 147 is cut off. As a result, rotor 107 is driven only by main coil 145 to keep rotating in the forward direction. Eccentric rotation of sub-shaft 125 via connecting rod 131 reciprocates piston 129 in cylinder 117, so that compression work is done.

Lubricant 103 rises in centrifugal pump 133 due to centrifugal force generated by centrifugal pump 133, and is transported to a lower end of forward leading groove 137, then transported to annular lubricant groove 141 by pumping force of forward leading groove 137.

The lubricant transported in annular lubricant groove 141 is pushed to the outer rim section of annular lubricant groove 141 by the centrifugal force, and raised through vertical hole 143 communicating with annular lubricant groove 141, thereby lubricating sliding sections such as connecting rod 131 and piston 129. Parts of the lubricant are discharged from an upper end of vertical hole 143 into a space of hermetic container 101. Since vertical hole 143 slants as shown in Fig. 3, centrifugal force is additionally added to the lubricant, so that an amount of the lubricant increases.

At this moment, if the lubricant flows into reverse leading groove 139, the lubricant is pushed down by downward force of reverse leading groove 139; however reverse leading groove 139 opens into inner rim of annular lubricant groove 141, and the lubricant is pushed to the outer rim of annular lubricant

groove 141 by the centrifugal force, so that little amount of the lubricant flows into reverse leading groove 139.

As shown in Fig. 3, reverse leading groove 139 never crosses with forward leading groove 137, so that the lubricant is hardly pushed down by reverse leading groove 139.

Further, because reverse leading groove 139 has a cross-sectional area smaller than that of forward leading groove 137, and reverse leading groove 139 has a lead greater than that of forward leading groove 137, the down-force generated by reverse leading groove 139 is so small that lubrication similar to the prior art can be maintained when the motor rotates in the forward direction.

Next, an operation of the compressor when the motor rotates in the reverse direction is explained. When the motor once stops, it is necessary to cool the PTC relay 149 in order to lower the resistance of elements of PTC relay 149 before the power is turned on again. If the time for cooling is too short, a turning-on of the power (e.g. just after an instantaneous power failure) does not allow a current to run through starting coil 147 because the elements of PTC relay 149 still have high resistance, so that the motor fails to start. In this case, if piston 129 is pushed back by repulsion force of compressed gas, and rotates the shaft in the reverse direction, the motor starts rotating in the reverse direction.

Centrifugal pump 133 produces pumping force regardless of a rotating direction, and lubricant 103 is transported to reverse leading groove 139 via centrifugal pump 133, forward leading groove 137 and thinner section 135. The lubricant transported to reverse leading groove 139 is transported to annular lubricant groove 141 by the pumping force of reverse leading groove 139.

The lubricant transported in annular lubricant groove 141 is pushed to the outer rim of annular lubricant groove 141 by the centrifugal force, and raised into vertical hole 143 communicating with annular lubricant groove 141, thereby lubricating sliding sections such as connecting rod 131 and piston 129.

5 Parts of the lubricant are discharged from an upper end of vertical hole 143 into a space of hermetic container 101. Since vertical hole 143 slants as shown in Fig. 3, centrifugal force is additionally added to the lubricant, so that an amount of the lubricant increases.

At this moment, if the lubricant flows into forward leading groove 137,

10 the lubricant is pushed down by downward force of forward leading groove 137; however forward leading groove 137 opens into inner rim of annular lubricant groove 141, and the lubricant is pushed to the outer rim of annular lubricant groove 141 by the centrifugal force, so that little amount of the lubricant flows into forward leading groove 137.

15 As shown in Fig. 3, forward leading groove 137 never crosses with reverse leading groove 139, so that the lubricant is hardly pushed down by forward leading groove 137.

Further, since reverse leading groove 139 has the cross-sectional area smaller than that of forward leading groove 137, and reverse leading groove 139

20 has a lead greater than that of forward leading groove 137, the pumping force generated by reverse leading groove 139 is so small that an amount of lubricant is smaller in the reverse rotation than in the forward rotation. Experiments tell that an amount of lubricant in the reverse rotation is approx. 20% as little as that in the forward rotation; however, this amount is enough for an operation

25 in several hours.

As discussed above, the lubricating mechanism of the present invention supplies a similar amount of lubricant to that of conventional ones when the

motor rotates in the forward direction, and supplies an amount enough to an operation in several hours when the motor rotates in the reverse direction. As a result, a compressor with high reliability is obtainable.

5 Industrial Applicability

The electric compressor of the present invention allows maintaining lubrication even in a reverse rotating operation, so that a highly reliable compressor is obtainable. The compressor can be used in vending machines and air-conditioners in addition to refrigerators.

CLAIMS

1. An electric compressor comprising:

a single-phase induction motor formed of a stator and a rotor;

5 a compressing mechanism driven by the motor; and

a hermetic container for accommodating the motor and the compressing mechanism and for pooling lubricant,

wherein the compressing mechanism includes:

a shaft having a main shaft and a sub-shaft;

10 a cylinder for forming a compressing chamber; and

a bearing for supporting the main shaft,

wherein the shaft includes:

a centrifugal pump opening into the lubricant;

15 a forward leading groove engraved on an outer wall of the main shaft, and having a first end communicating with the centrifugal pump and a second end communicating with an annular lubricant groove provided on an upper end of the bearing;

20 a reverse leading groove having a lead directing in an opposite direction to that of the forward leading groove, a first end communicating with the centrifugal pump, and a second end directly opening to the annular lubricant groove; and

a vertical hole bored in the sub-shaft and having a first end communicating with the annular lubricant groove, and a second end opening into the hermetic container.

25

2. The electric compressor of claim 1, wherein the reverse leading groove of which first end communicates with the centrifugal pump via a thinner

section formed at an intermediate section of the shaft.

3. The electric compressor of claim 1 or 2, wherein a cross sectional area of the reverse leading groove is smaller than that of the forward leading groove.

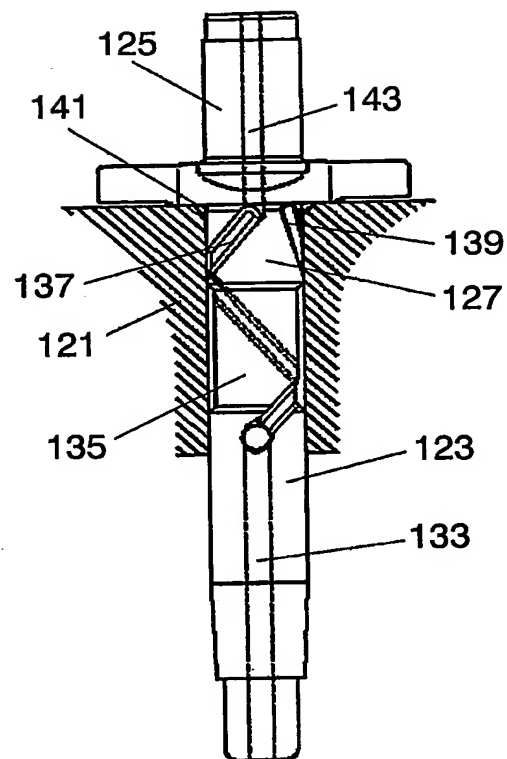
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4. The electric compressor of claim 1 or 2, wherein a lead of the reverse leading groove is greater than that of the forward leading groove.

5. The electric compressor of claim 1, wherein the vertical hole slants
10 with respect to a shaft center of the main shaft such that an upper section of the vertical hole slants outward.

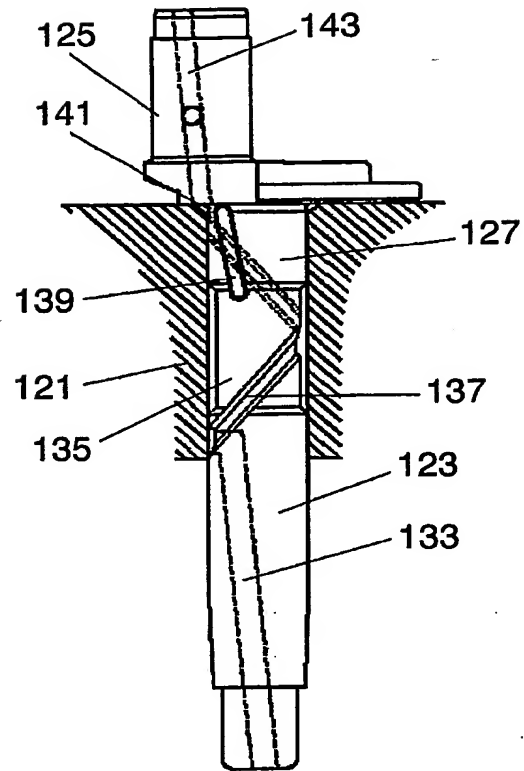
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FIG. 2



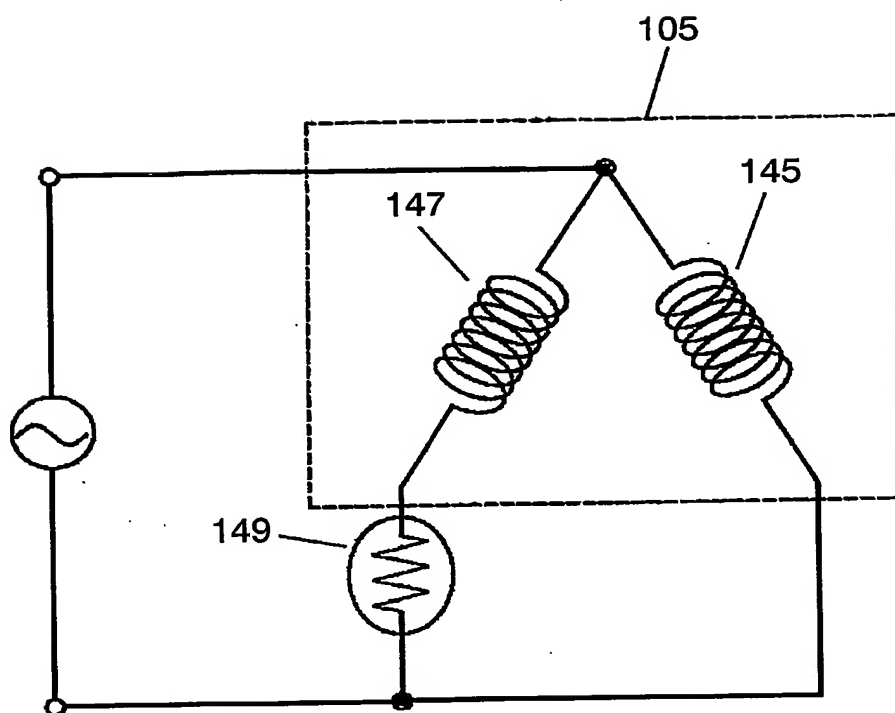
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FIG. 3



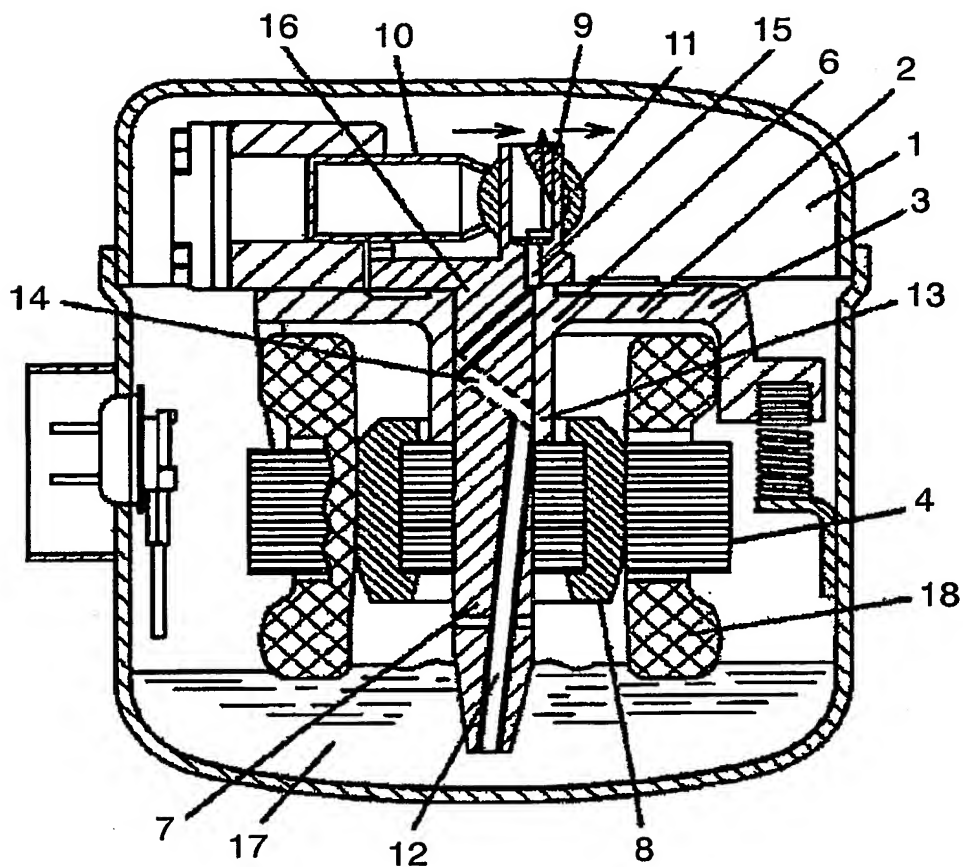
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FIG. 4



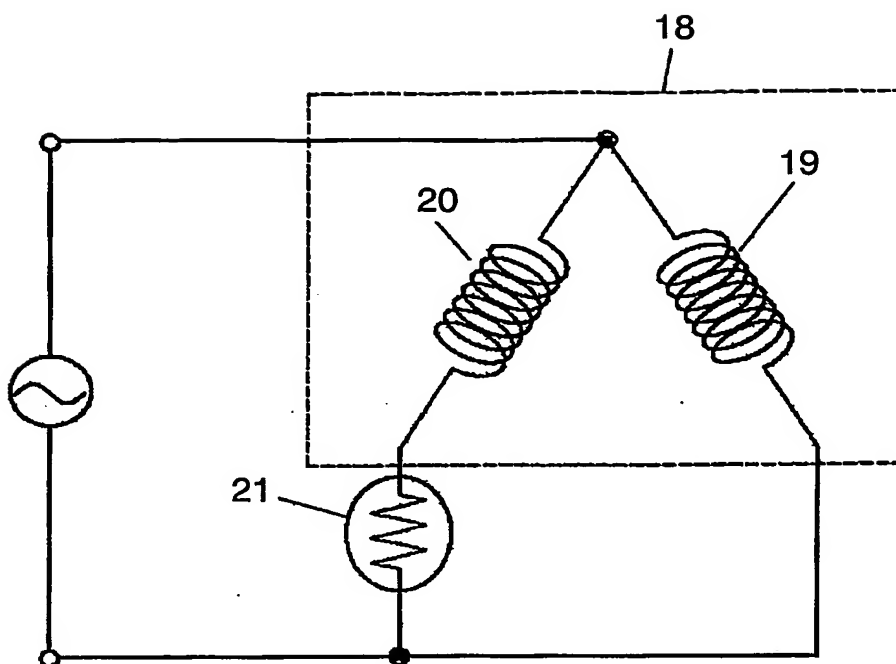
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FIG. 5



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FIG. 6



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Reference numerals in the drawings

- 101 hermetic container
- 103 lubricant
- 105 stator
- 107 rotor
- 109 single-phase induction motor
- 111 compressing mechanism
- 117 cylinder
- 121 bearing
- 123 main shaft
- 125 sub-shaft
- 127 shaft
- 133 centrifugal pump
- 135 thinner section
- 137 forward leading groove
- 139 reverse leading groove
- 141 annular lubricant groove
- 143 vertical hole

INTERNATIONAL SEARCH REPORT

International Application No

PCT/JP2005/007359

A. CLASSIFICATION OF SUBJECT MATTER

IPC 7 F04B39/02

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

IPC 7 F04B

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the International search (name of data base and, where practical, search terms used)

EPO-Internal, PAJ

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category *	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
A	GB 1 023 018 A (VEB DKK SCHARFENSTEIN) 16 March 1966 (1966-03-16) figure 1	1,5
A	----- WO 03/052271 A (LG ELECTRONICS INC; PARK, KYOUNG JUN; KIM, KEE JOO; KIM, HEE HYUN; KIM) 26 June 2003 (2003-06-26) abstract; figures page 4, lines 11-24 page 12, lines 15-18 claim 1	1,5
A	----- PATENT ABSTRACTS OF JAPAN vol. 2000, no. 06, 22 September 2000 (2000-09-22) & JP 2000 087856 A (MATSUSHITA REFRIG CO LTD), 28 March 2000 (2000-03-28) abstract ----- -/-	1



Further documents are listed in the continuation of box C:



Patent family members are listed in annex.

* Special categories of cited documents:

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O document referring to an oral disclosure, use, exhibition or other means

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Date of the actual completion of the International search

7 July 2005

Date of mailing of the International search report

13/07/2005

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INTERNATIONAL SEARCH REPORT

International Application No

PCT/JP2005/007359

C.(Continuation) DOCUMENTS CONSIDERED TO BE RELEVANT

Category *	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
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Information on patent family members

International Application No

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